

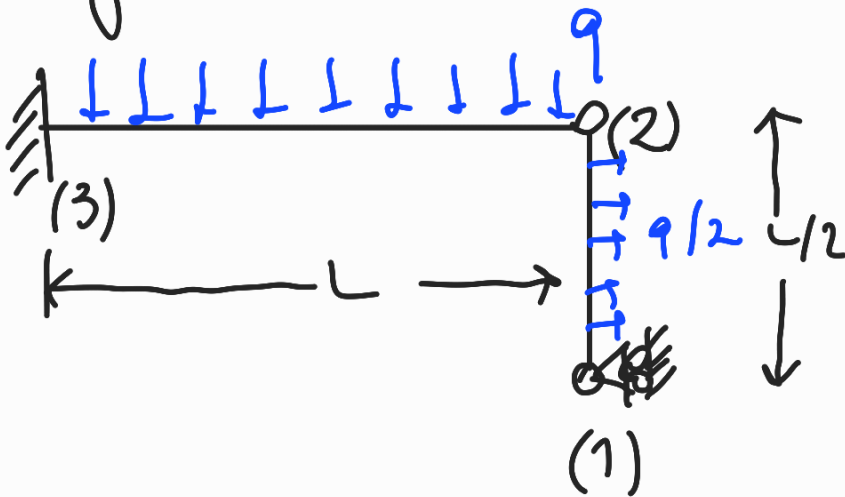
1) • Verifica che la struttura è isostatica

$$n_{aste} = 2$$

$$v = 3 + 2 + 1 = 6$$

$$g = 2 \cdot 3 = 6$$

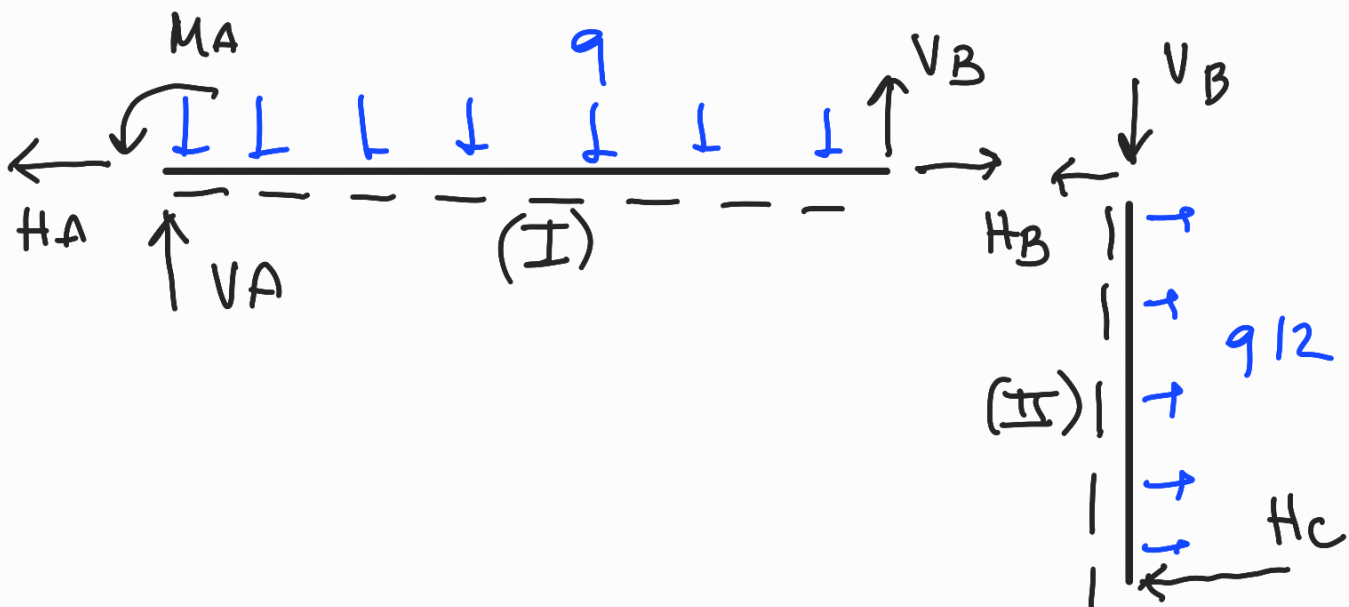
→ isostatica



$$L = 6 \text{ m}$$

$$q = 12000 \text{ N/m}$$

• Calcolo delle reazioni vincolari



Equilibri delle aste;

$$I \begin{cases} \rightarrow^+ & -H_A + H_B = 0 \\ \uparrow & V_A + \cancel{V_B} - qL = 0 \\ \curvearrowright_A & -M_A + q \frac{L^2}{2} - \cancel{V_B} \cdot L = 0 \end{cases}$$

$$H_A = H_B = 9000 \text{ N}$$

$$V_A = qL = 72000 \text{ N}$$

$$M_A = \frac{qL^2}{2} = 216000 \text{ Nm}$$

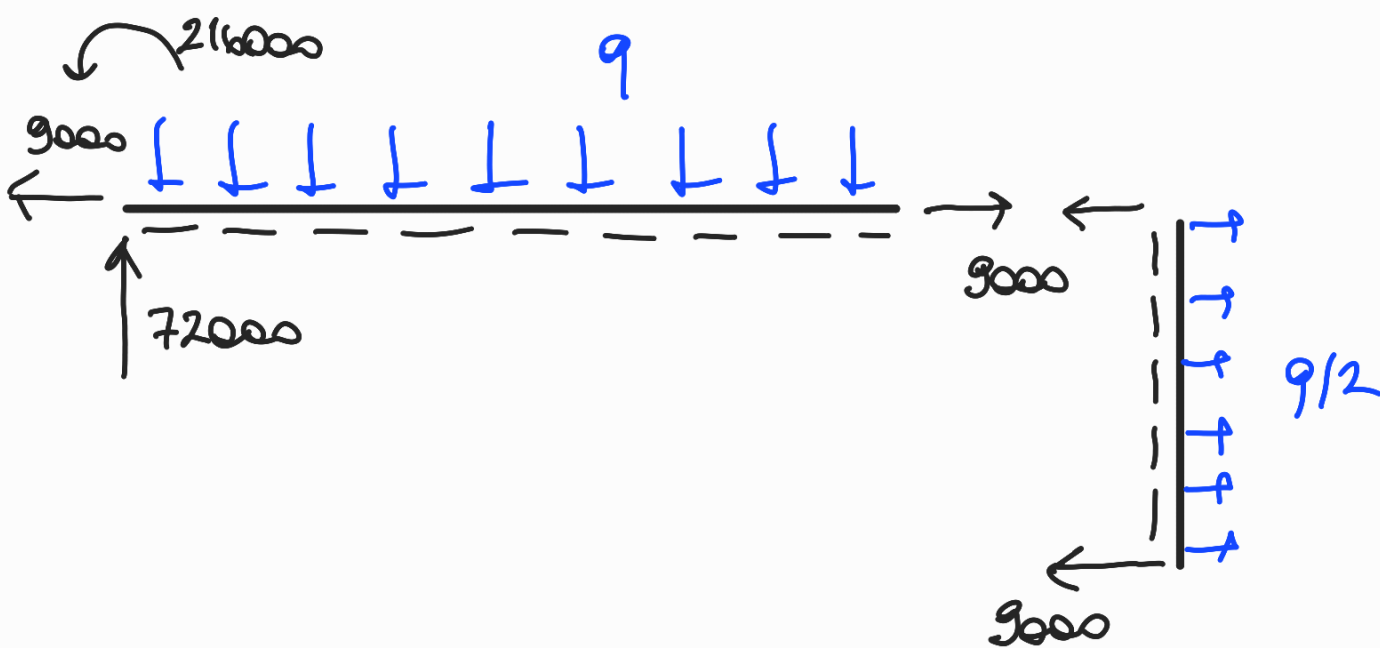
$$II \begin{cases} \rightarrow^+ & -H_B + \frac{q}{2} \cdot \frac{L}{2} - H_C = 0 \\ \uparrow & -V_B = 0 \\ \curvearrowright_B & -\frac{q}{2} \frac{L}{2} \frac{L}{4} + H_C \cdot \frac{L}{2} = 0 \end{cases}$$

$$H_B = \frac{qL}{4} - H_C = 9000$$

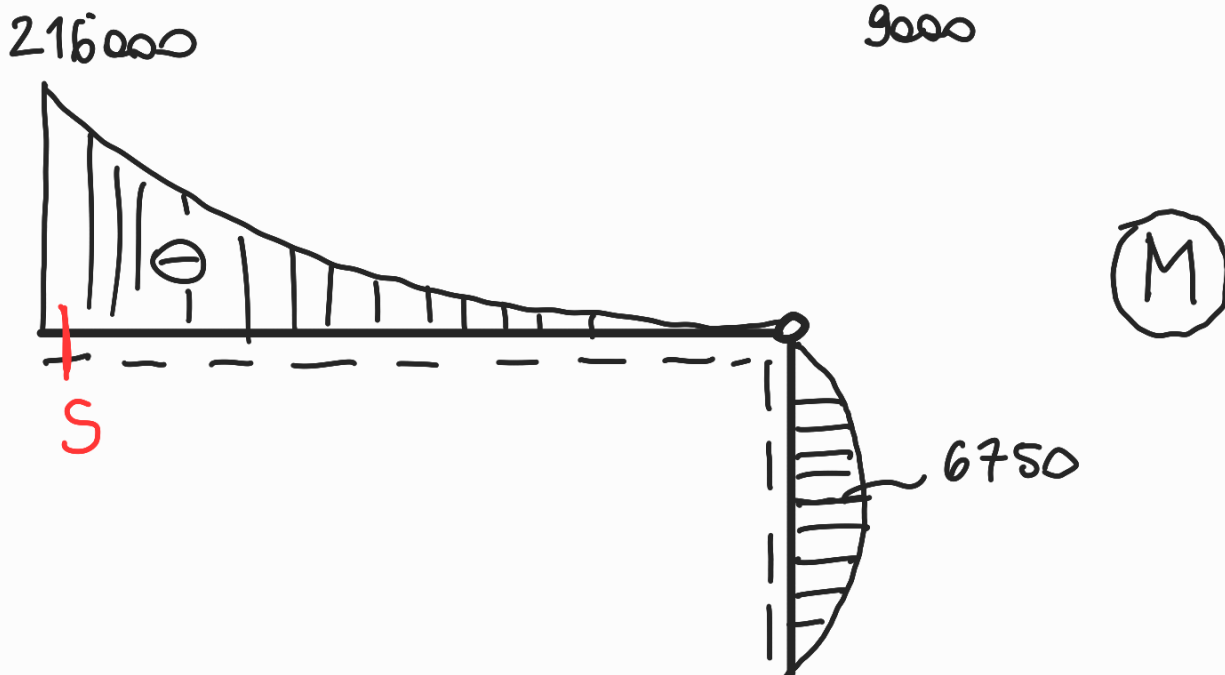
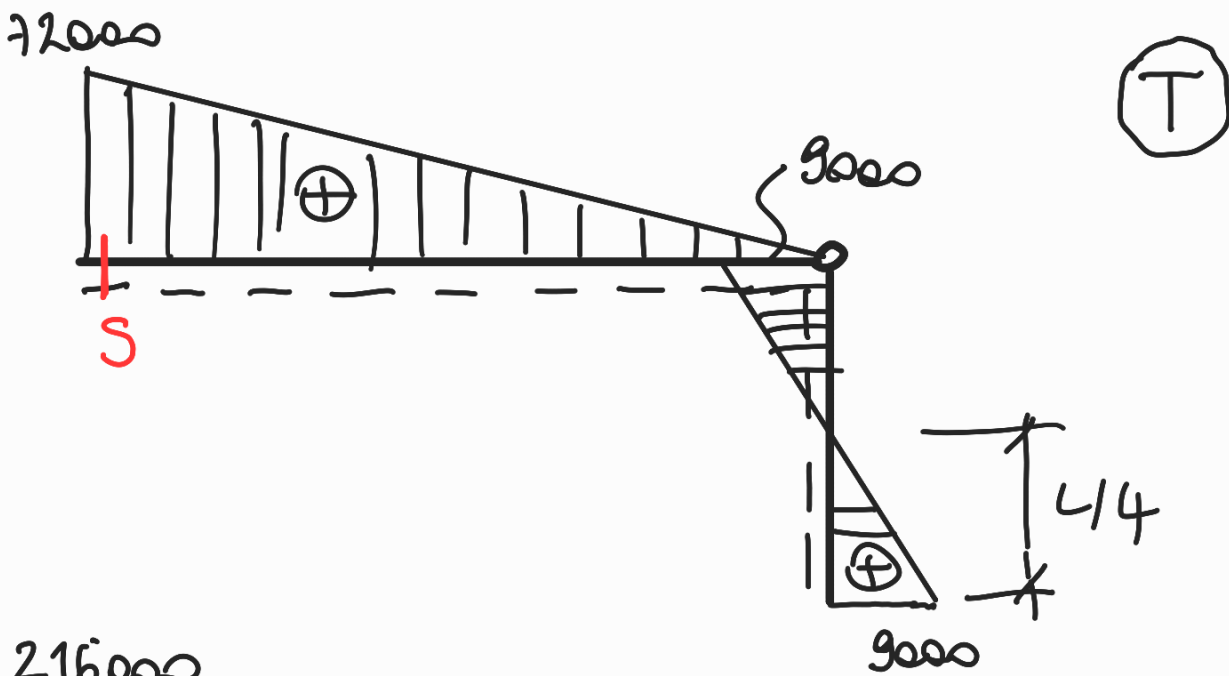
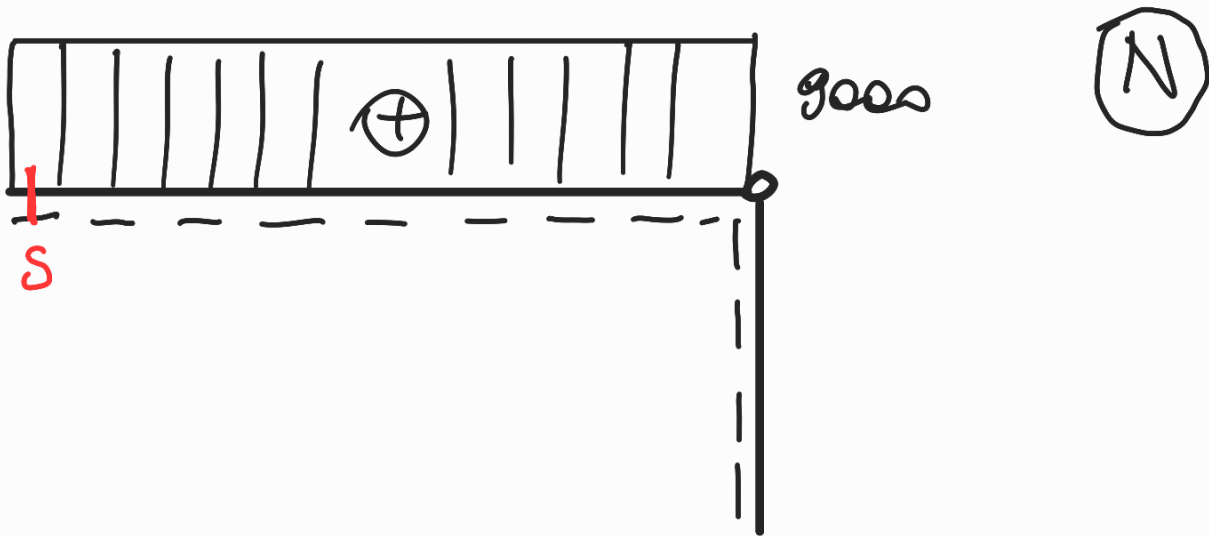
$$V_B = 0 \text{ N}$$

$$H_C = \frac{qL}{8} = 9000 \text{ N}$$

Schema finale delle forze attive e reattive

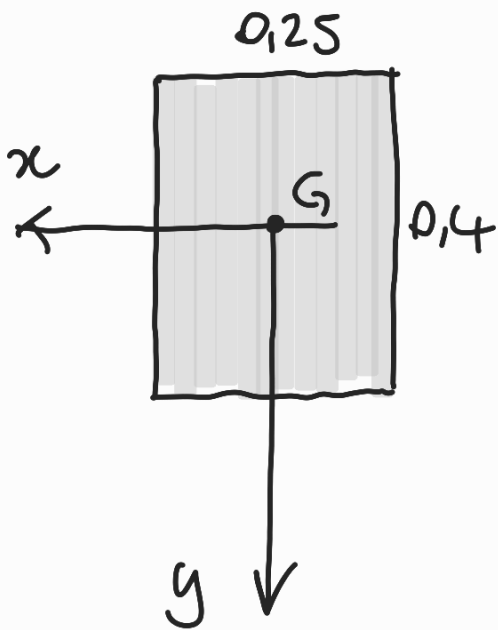


• Diagrammi delle azioni interne



2)

- Caratteristiche geometriche della sezione

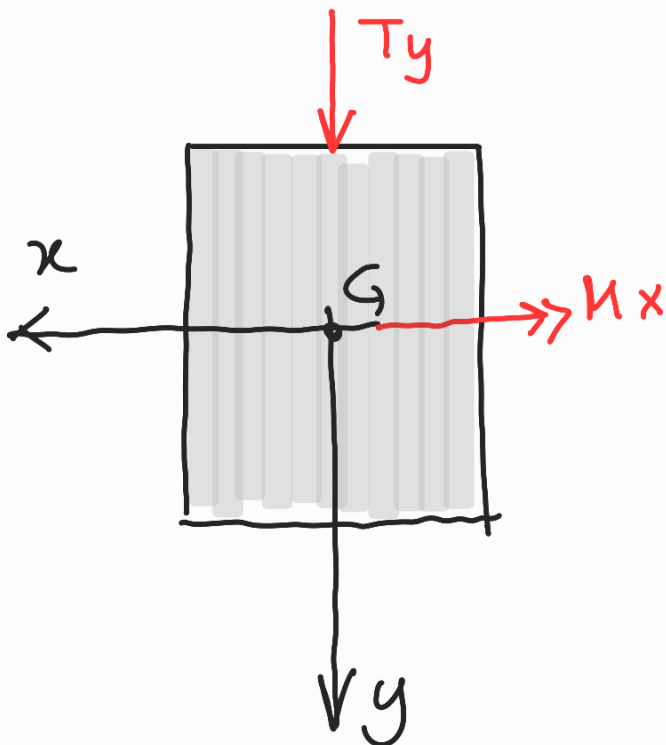


$$A = 0,25 \cdot 0,4 = 0,1 \text{ m}^2$$

$$I_x = \frac{0,25 \cdot 0,4^3}{12} = 1,33 \cdot 10^{-3} \text{ m}^4$$

Il momento di inerzia I_y non è necessario.

- Calcolo delle tensioni nelle sez. S

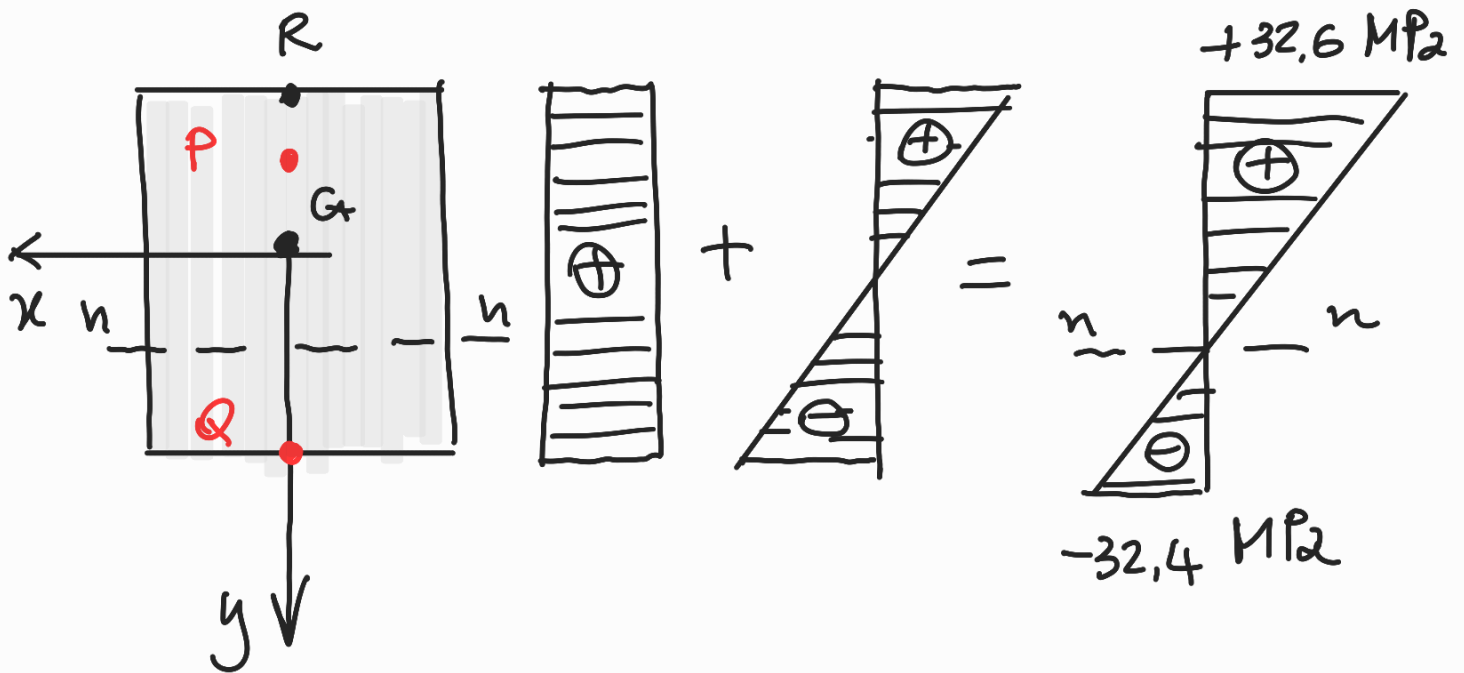


$$N = 9000 \text{ N}$$

$$T = 72000 \text{ N}$$

$$M_x = 216000 \text{ Nm}$$

• Sforzo normale eccentrico



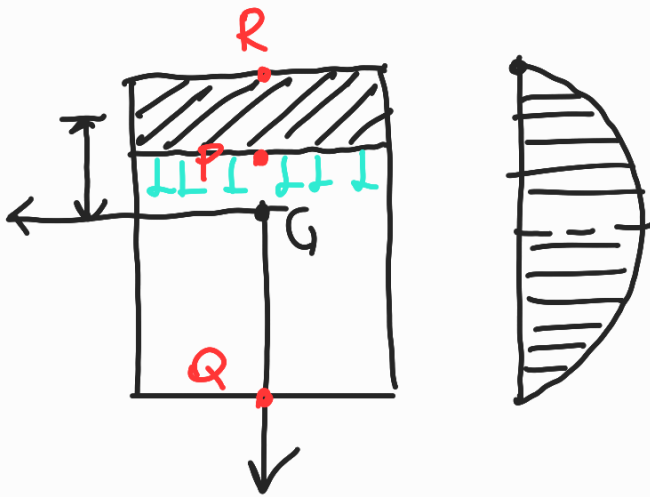
$$\sigma_z(y) = \frac{N}{A} + \frac{M_x}{I_x} \cdot y$$

$$\begin{aligned} \sigma_z(R) &= \frac{9000}{0,1} + \frac{-216000}{1,3 \cdot 10^{-3}} \cdot (-0,2) = 90000 + 3,25 \cdot 10^7 \\ &= 3,26 \cdot 10^7 \\ &= 32,6 \text{ MPa} \end{aligned}$$

$$\begin{aligned} \sigma_z(P) &= \frac{9000}{0,1} - \frac{216000}{1,3 \cdot 10^{-3}} \cdot (-0,1) = 9000 + 1,625 \cdot 10^7 \\ &= 1,634 \cdot 10^7 \text{ Pa} \\ &= 16,34 \text{ MPa} \end{aligned}$$

$$\begin{aligned} \sigma_z(Q) &= \frac{9000}{0,1} - \frac{216000}{1,3 \cdot 10^{-3}} \cdot (0,2) = 90000 - 3,25 \cdot 10^7 \\ &= -3,24 \cdot 10^7 \text{ Pa} \\ &= -32,4 \text{ MPa} \end{aligned}$$

- Taglio (f. di Jourawski) $\tau_{zs} = -\frac{T_y S_x^*}{I_x \cdot b}$



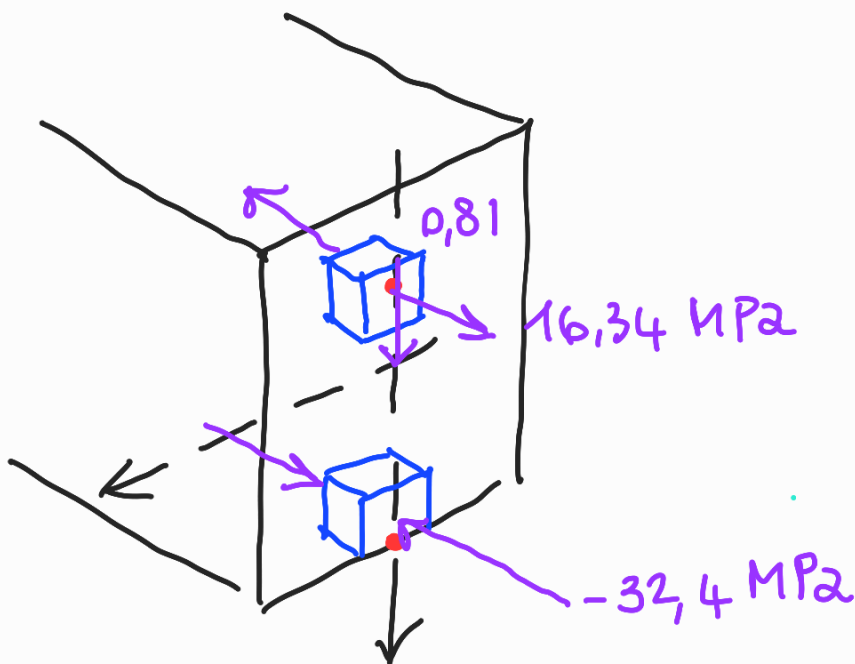
$$\tau_{zy} = \frac{-72000 \cdot S_x^*}{1,33 \cdot 10^{-3} \cdot 0,25}$$

$$\tau_{zy}(P) = \frac{-72000 \cdot (-0,25 \cdot 0,1) \cdot 0,15}{1,33 \cdot 10^{-3} \cdot 0,25} = 0,812 \text{ MPa}$$

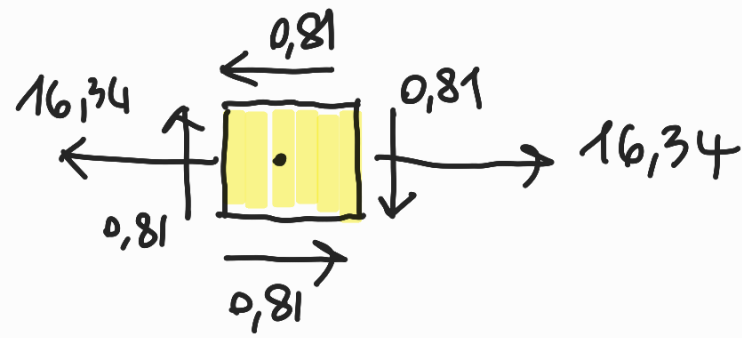
$$\tau_{zy}(G) = \frac{-72000 \cdot (-0,25 \cdot 0,2 \cdot 0,1)}{1,33 \cdot 10^{-3} \cdot 0,25} = 1,08 \text{ MPa}$$

$$\tau_{zy}(Q) = 0$$

- Verifica di resistenza nel punto Q delle sezioni S e cerchio di Mohr

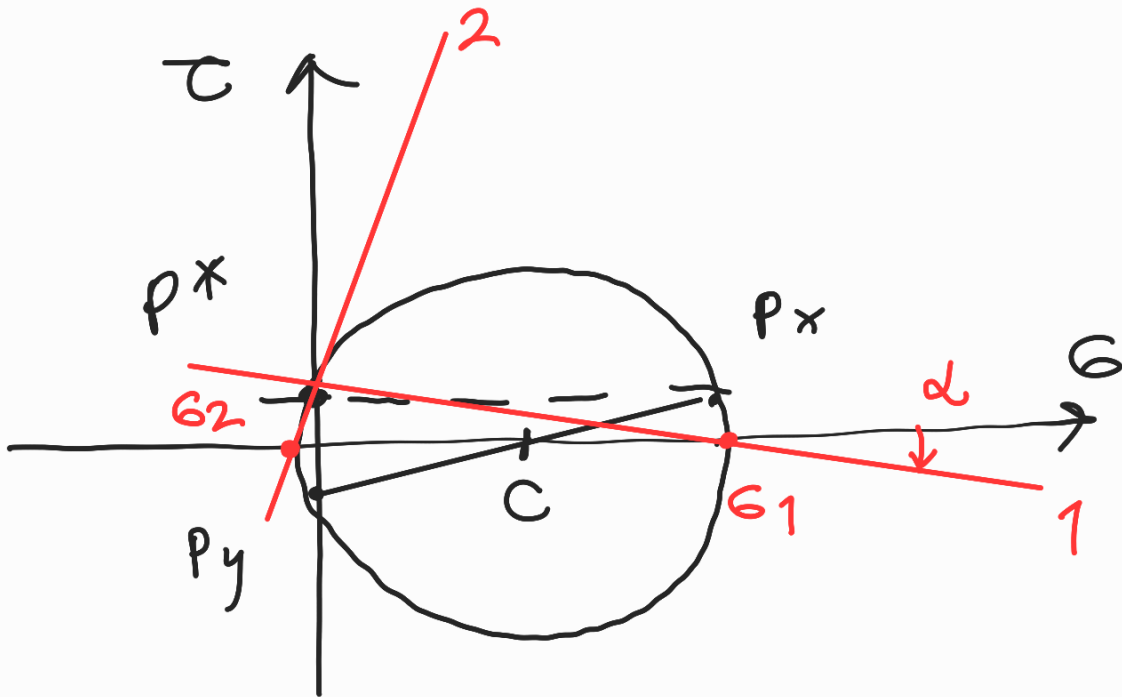


Punto P



$$P_x(16,34; 0,81)$$

$$P_y(0; -0,81)$$

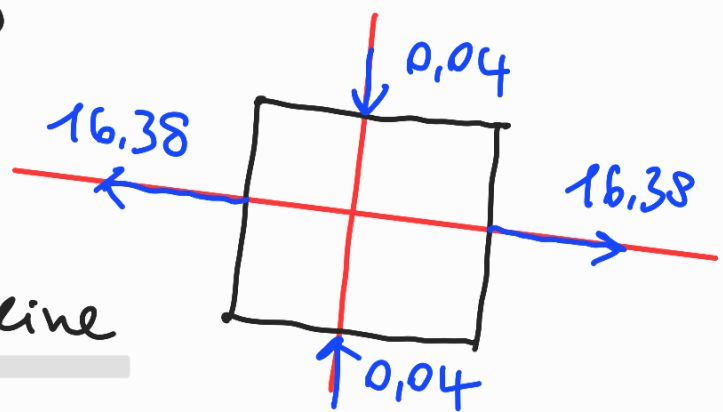


$$\sigma_c = 8,17 \text{ MPa} \quad r = 8,21 \text{ MPa}$$

$$\sigma_1 = \sigma_c + r = 16,38 \text{ MPa}$$

$$\sigma_2 = \sigma_c - r = -0,04 \text{ MPa}$$

$$\alpha = \frac{1}{2} \arctg \frac{\tau_{xy}}{\frac{\sigma_x - \sigma_y}{2}} = -2^{\circ},83 \text{ orario}$$



Verifica con crit. di Rankine

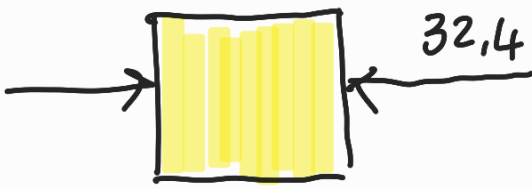
$$\sigma_{c, 2mm} = 20 \text{ MPa}$$

$$\sigma_{t, 2mm} = 8 \text{ MPa}$$

$$\sigma_1 = 16,38 < 20 \text{ verificato}$$

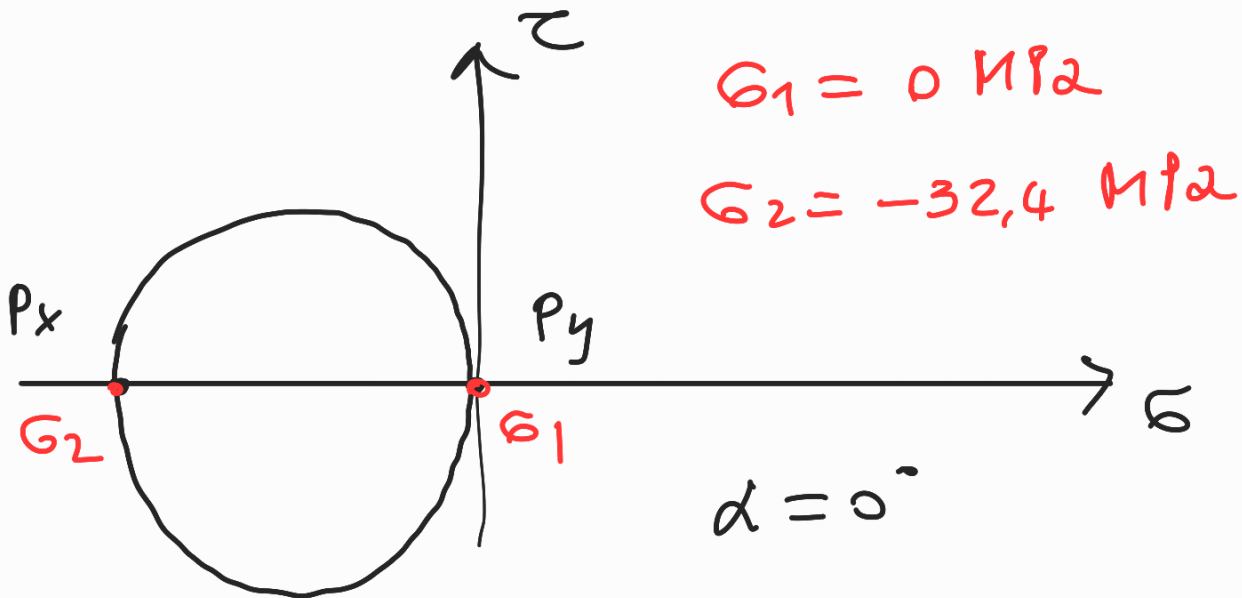
$$\sigma_2 = |-0,04| < 8 \quad \text{"}$$

Punto Q



$$P_x (-32,4; 0)$$

$$P_y (0; 0)$$



Verifica

$$\sigma_1 = 0 < 8 \text{ verificato}$$

$$\sigma_2 = |-32,4| > 20 \text{ non verificato}$$

